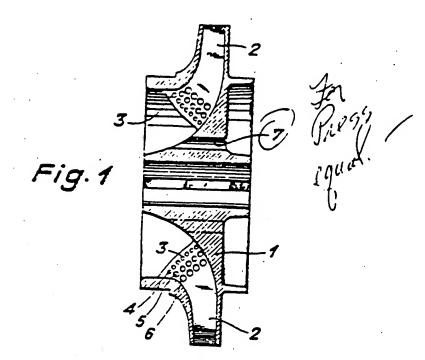
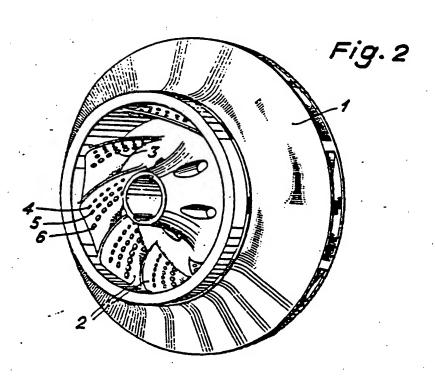
# 942.648

342.648 COMPLETE SPECIFICATION

This drawing is a reproduction of the Original on a reduced scale.





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## PATENT SPECIFICATION

DRAWINGS ATTACHED

942,648

Date of Application and filing Complete Specification June 25, 1962. No. 24:33 62.

Application made in Switzerland (No. 7,443) on June 27, 1961. Complete Specification Published Nov. 27, 1963.

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SCIENTIFIC LIBRARY DEC 3 0 1963 U. S. PATENT OFFICE

index of acceptance: - Class F1 C2F1. Interacional Classification:-F 05 c.

## COMPLETE SPECIFICATION

## Centrifugal Pumps

We, SULZER FRÈRES, SOCIÉTÉ ANONYME, a Company organised under the Laws of Switzerland, cf Winterthur, Switzerland, do hereby declare the invention, for which we pray that a parent may be granted to us, and the method by which it is to be parformed, to be particularly described in and This invention relates

invention relates to centrifugal pumps.

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Where it is required to convey large quantitles of liquid, as with condensates in steam power stations, it is preferred to use a centrifugal pump having a large inlet. With such 15 a pump there may be considerable pressure surges in some conditions of operation. Quite apart from the fact that such surges upset normal operation, they should be suppressed in order to prevent pumps and pipes from being damaged. The surges occur when, for a given delivery adjusted by a flow controller in the delivery pipe, the absolute value of the inlet head of the liquid is in a critical range. If the delivery is altered by adjust-25 ing the controller, the pressure surges are produced at a different inlet head. These inler heads are always several times greater than the inlet heads at which cavitation tpenomens occur.

According to the invention, the biades of forzuons near their leading edges.

30 . Previously, such surges have been suppressed by very rapid alteration of the delivery in order to effect a shift away from the critical range, because it is usually impos-sible to alter the inlet head rapidly enough. The disadvantage of this rapid alteration of the delivery is that the delivery is affected without regard to any other kind of delivery control, for instance, in dependence on an operational variable.

the rotor of the first (or only) stage of the pump are provided with a number of per-

At inlet heads at which pressure surges would occur in the absence of apertures, no [Price 4s. 6d.]

sures octur when the perforations are present. The control range over which normal operation can be provided is thus advantageously increased; indeed, the invention may in some cases enable surging to be obviated completely. Conditions in respect of cavitation are not affected; in particular, cavitation does not occur any earlier than would be the case with no perforations. Also, there is only a small decrease in efficiency.

The perforations can be disposed perpendicularly to the blade surface irregularly or in rows parallel to the leading edge; the diameters of the perforations can be equal to or different from one another, while the perforations in a row may be equally or unequaliy spaced. Where there are three or more rows of perforations the spacings between the rows may be equal or unequal. The diameters of the perforations, at least in the first rows, can increase from front to rear. In the case of multi-stage centrifugal pumps, the surges can be obviated if the blades of the rotor of only the first stage are perforated.

The invention may be performed in various ways and a specific embediment will now be described by way of example with reference to the accompanying drawings, in

Figure 1 is a longitudinal section through

the rotor; and
Figure 2 is a perspective view of the rotor. Referring to the drawings, the rotor 1 of a centrifugal pump has blades 2. blade 2 is provided with perforations 3 in its front part, i.e. near the leading edge. In the embodiment illustrated, the perforations 3 are arranged in three rows 4-6 which extend parallel with the leading edge of each The diameters of the perforations increase from row to row in the direction of flow, being larger in row 5 than in row 4 and larger in row 6 than in row 5. Conventional passages 7 extending parallel with

the rotor axis are provided in the rotor to equalise the pressure on either side thereof. The rotor illustrated in the drawings can be used for a single-stage pump or as rotor of the first stage of a multi-stage pump.

WHAT WE CLAIM IS:

 A centrifugal pump in which the blades of the rotor of the first (or only) stage are provided with a number of perforations near their leading edges.

2. A centrifugal pump as claimed in Claim
1 in which the perforations are disposed in
2 at least one row.

3. A centrifugal pump as claimed in Claim
15 2 in which the diameters of all the perforations in a row are equal.

4. A centrifugal pump as claimed in Claim
2 in which there are at least two rows of
perforations and the diameters of the per20 forations of the various rows are of different

A centrifugal pump as claimed in Claim
 in which the diameters of the perforations

in at least the leading rows increase in size

6. A centrifugal pump as claimed in any of Caims 2 to 5 in which the perforations in a row are equally spaced.

7. A committingal pump as claimed in any of Claims 2 to 5 in which the perforations in a row are unequally spaced.

8. A centrifugal pump as claimed in any of Claims 2 to 7 in which there are at least three rows of perforations and the spacings between the rows are equal.

9. A centrifugal pump as claimed in any of Claims 2 to 7 in which there are at least three rows of perforations and the spacings between the rows are unequal.

10. A centrifugal pump substantially as described with reference to the accompanying drawings.

KILBURN & STRODE, Chartered Patent Agents, Agents for the Applicants.

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Learnington Spa: Printed for Her Majesty's Stationery Office by the Courier Press.—1963.

Published at The Patent Office, 25, Southampton Buildings, London, W.C.7, from which copies may be obtained.